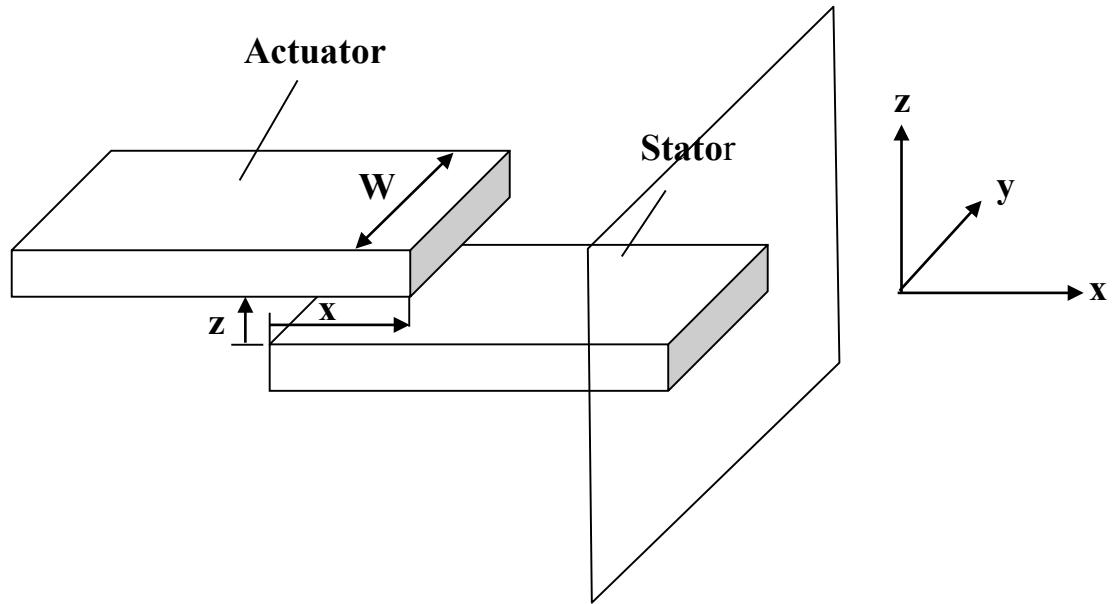


## Lecture 8-2 Design Comb-drive Actuator through

### Cronos/MUMPS Process: I

#### ◆ Electrostatic actuator/sensor [1]



#### 1. Fundamental

##### a. Energy in a battery-capacitor system

$$U_{c1} = VQ \quad (8-1)$$

$$\begin{aligned} U_{c1} &= VQ \\ U_{c2} &= U_{c2,b} + U_{c2,c} \\ &= V(Q - q) + \int_0^q Vdq \end{aligned}$$

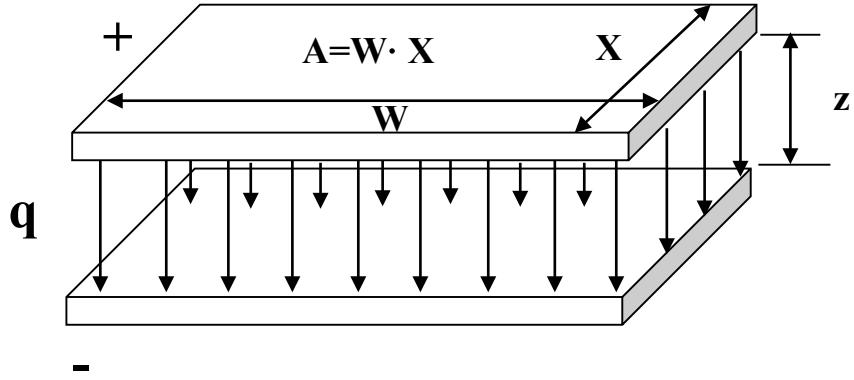
Energy stored in capacitance

$$= V(Q - CV) + \frac{1}{2}CV^2 \quad (q = CV) \quad (8-2)$$

$$\begin{aligned} U_{c2} &= VQ - \frac{1}{2}CV^2 \quad (8-3) \\ (Q-q)C &= VQ - \frac{1}{2}CV^2 \end{aligned}$$

System energy reduced by  $\frac{1}{2}CV^2$

### b. Capacitance of a flat plate



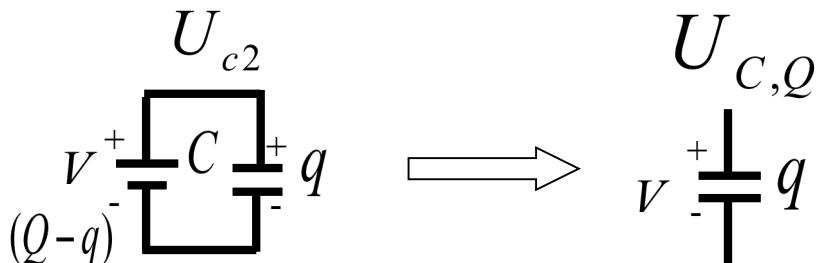
**Neglect end effect**

$$C = k \frac{\epsilon_0 A}{z} = k \frac{\epsilon_0 W X}{z} \quad (8-4)$$

Here  $k$ : Dielectric constant, for vacuum: 1.0, air: 1.00059,  
Pyrex: 5.6, Teflon: 2.1, water 80.

$\epsilon_0$ : Permittivity of free space:  $8.85 \times 10^{-12} \text{ C}^2/\text{Nm}^2$

### 2. Constant Charge mode



**Energy stored:**

$$U_c = \frac{1}{2} CV_0^2 = \frac{q_0^2}{2C} = \frac{1}{2k\epsilon_0} \left( \frac{z}{WX} \right) q_0^2 \quad (8-5)$$

**a. Fix gap z, move in X direction**

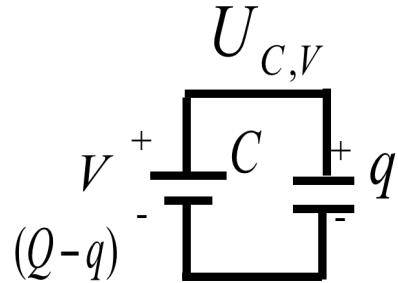
$$Q F_x = \frac{-\partial U_C}{\partial X} = \frac{1}{2k\epsilon_0} \left( \frac{z}{WX^2} \right) q_0^2 \quad (8-6)$$

**b. Fix X position, move in z direction**

$$Q F_z = \frac{-\partial U_C}{\partial z} = \frac{1}{2k\epsilon_0} \left( \frac{1}{WX} \right) q_0^2 \quad (8-7)$$

**Constant force!!**

**3. Constant Voltage mode**



**Energy stored:**

$$U_{c,V} = VQ_0 - \frac{1}{2} CV_0^2 = VQ_0 - \frac{1}{2} \left( \frac{k\epsilon_0 WX}{z} \right) V_0^2 \quad (8-8)$$

**a. Fix gap z, move in X direction**

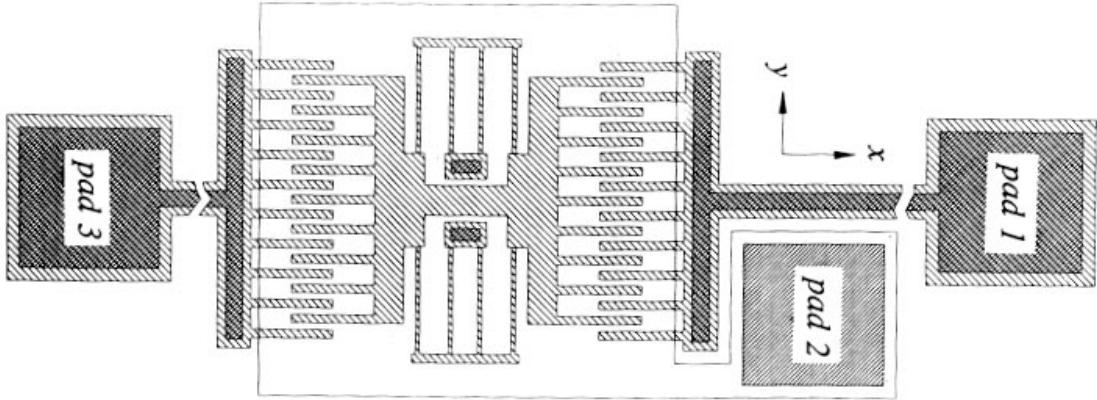
$$V F_x = \frac{-\partial U_C}{\partial X} = \frac{1}{2} \left( \frac{k\epsilon_0 W}{z} \right) V_0^2 \quad (8-9)$$

**Constant force!!**

**c. Fix X position, move in z direction**

$$_V F_z = \frac{-\partial U_C}{\partial z} = \frac{-1}{2} \left( \frac{k\epsilon_0 W X}{z^2} \right) V_0^2 \quad (8-10)$$

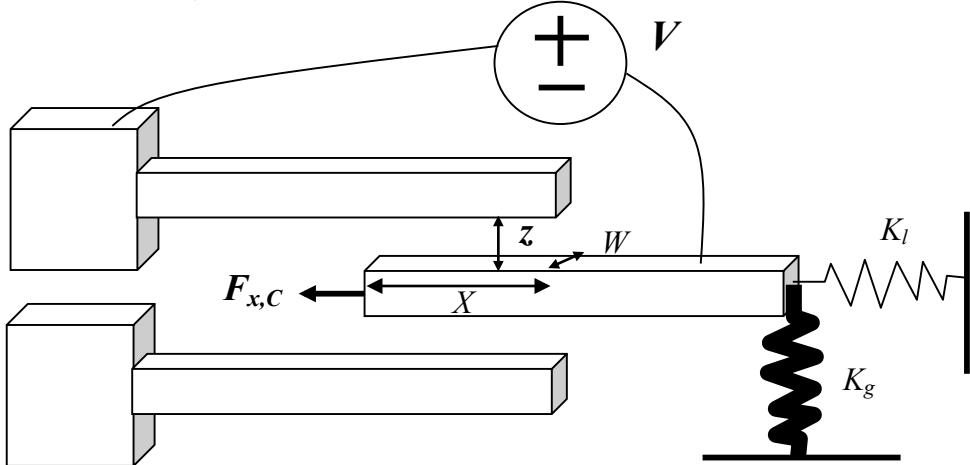
◆ Comb drive actuator (Tang et. Al., MEMS'89)[1]



Layout of a linear resonant plate with comb structures on both ends and a 50  $\mu\text{m}$  long folded-beam on each side.

### 1. Driving force and displacement

#### a. Laterally driven

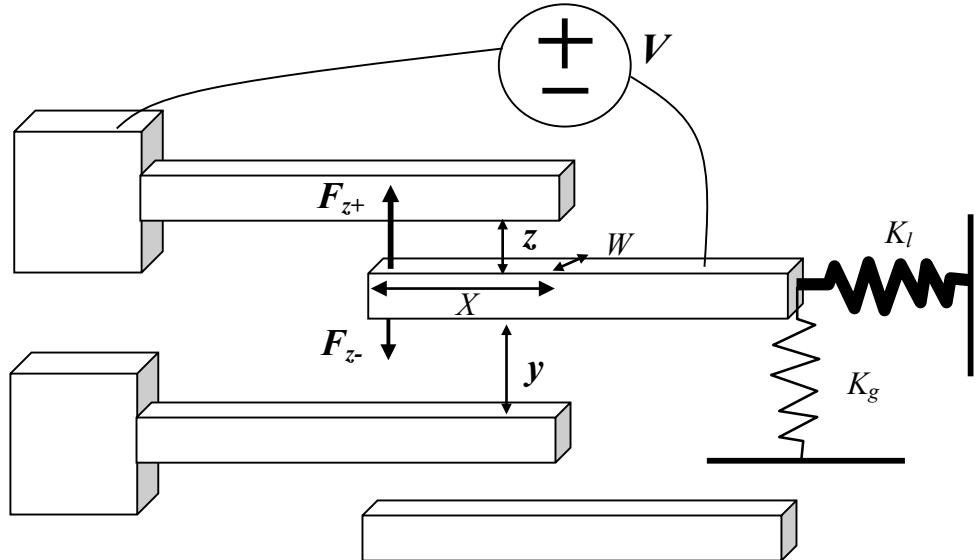


$$C_{total} = N\epsilon_0 \frac{WX}{z} \quad (\text{For } W \gg Z, N: \# \text{ of gaps}) \quad (8-11)$$

$$F_{x,C} = \frac{1}{2} N\epsilon_0 V^2 \frac{W}{z} \quad (8-12)$$

- i. Large displacement
- ii. Small force
- iii. Constant force

**b. Gap-closing driven**



$$C_{total,z} = N_z \epsilon_0 \frac{WX}{z} \quad (\text{For } W \gg z, \text{ } N_z: \# \text{ of closer gaps}) \quad (8-13)$$

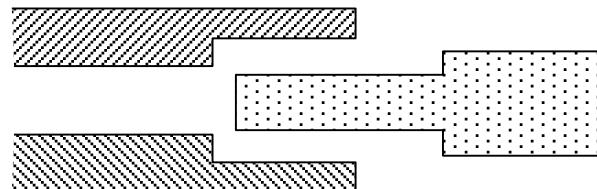
$$C_{total,y} = N_y \epsilon_0 \frac{WX}{y} \quad (\text{For } W \gg y, \text{ } N_y: \# \text{ of farer gaps}) \quad (8-14)$$

$$\begin{aligned} F_z &= F_{z+} - F_{z-} = \frac{-1}{2} N_z \left( \frac{k \epsilon_0 W X}{z^2} \right) V^2 - \frac{-1}{2} N_y \left( \frac{k \epsilon_0 W X}{y^2} \right) V^2 \\ &\approx \frac{-1}{2} N_z \left( \frac{k \epsilon_0 W X}{z^2} \right) V^2 \quad (for z > y) \end{aligned} \quad (8-15)$$

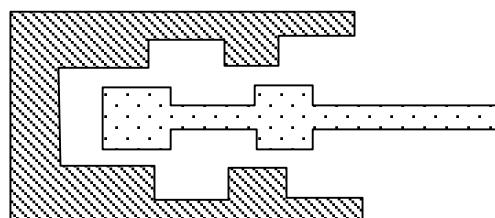
- i. Nonlinear force
- ii. Small displacement

## 2. To get larger force

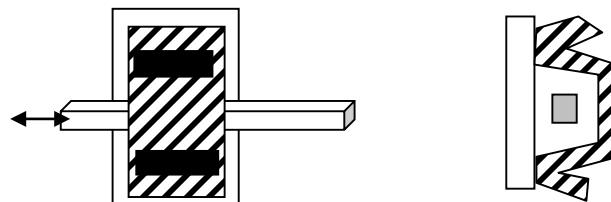
- a. Gap closer-better lithography resolution
- b. Higher aspect ratio- using LIGA, SCREAM, Deep Silicon RIE, Thick PR...increase W
- c. Special initial fixing technique [2]...decrease Z



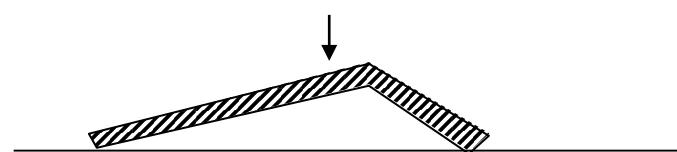
- d. Increase capacitive area: tooth like surface



- e. Increase capacitive area: Cylinder type surface

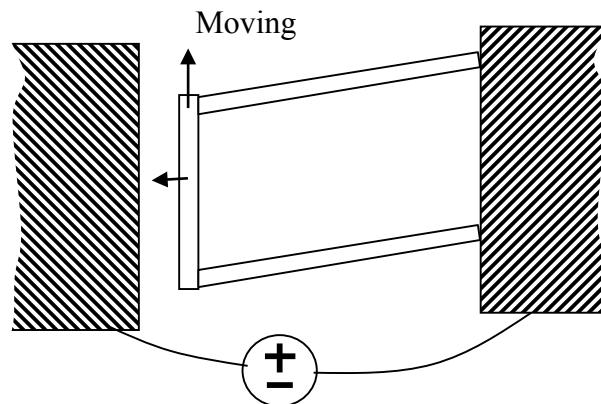


- f. Using buckling effect-scratch actuator

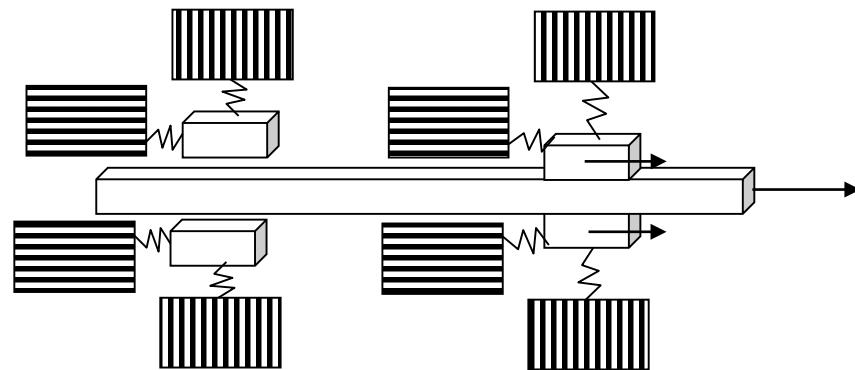


### 3. To get larger Displacement

#### a. Laterally driven+gap closing [3]

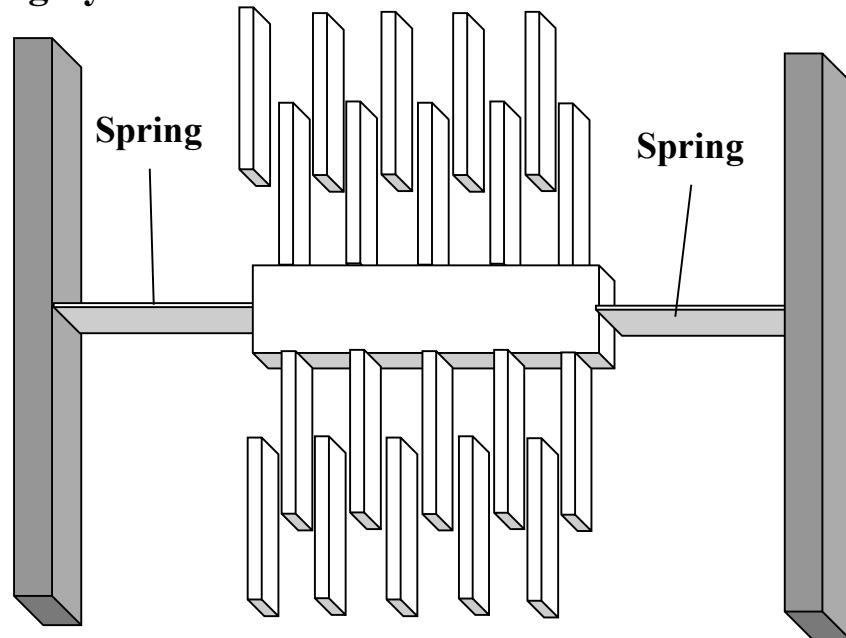


#### b. Inch worm

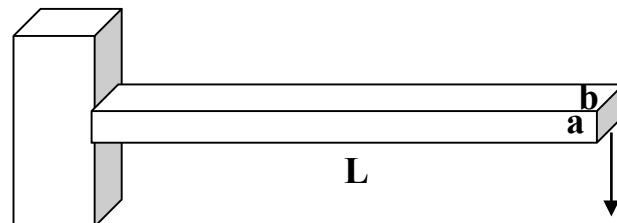


#### c. Transverse motion=>rotational motion (Sandia national lab, comb drive-> gear set)

## ◆ Spring System

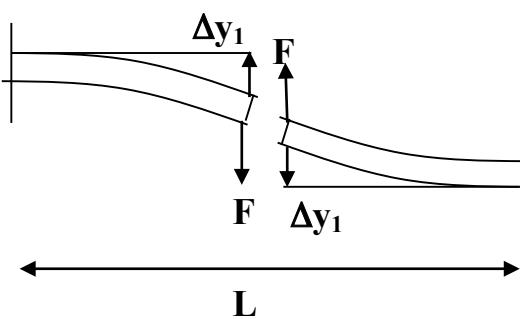


### 1. Cantilever beam



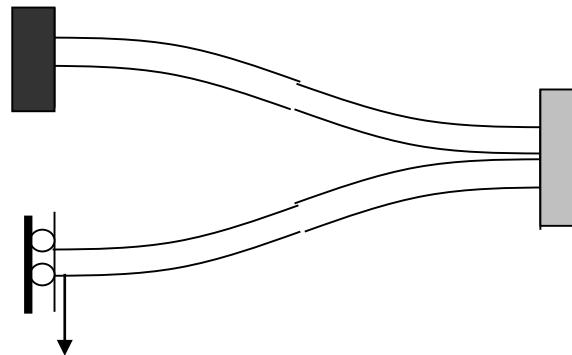
$$K(a,b,L) = \frac{F}{\Delta y} = \frac{Ea^3b}{4L^3} = K_0 \quad (8-16)$$

### 2. Two ends constrained beam:



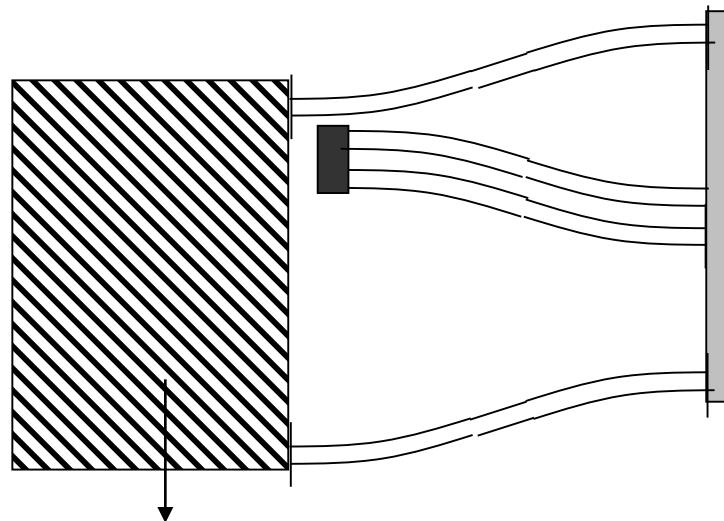
$$k_1 = \frac{F}{2\Delta y_1} = \frac{1}{2} k(a, b, \frac{L}{2}) = 4K_0 \quad (8-17)$$

### 3. Crab lag



$$k_2 = \frac{1}{2} K_1 = 2K_0 \quad (8-18)$$

### 4. Folded beam



$$k_{fold} = 2K_2 = 4K_0 \quad (8-19)$$

## ◆ Viscous Damping [4]

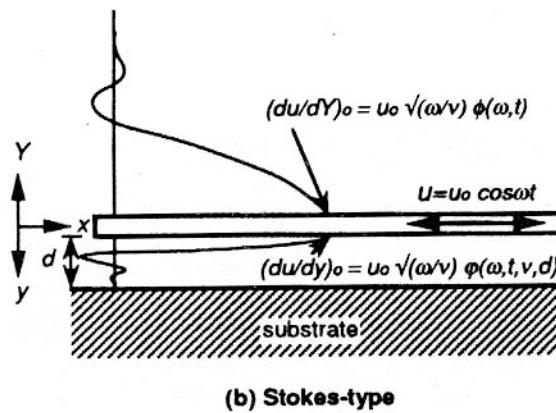
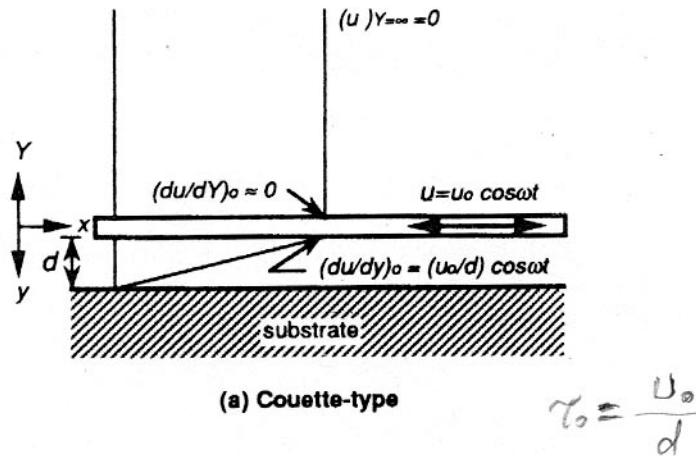


Fig. 2. Velocity profiles in the fluid layers induced by a laterally oscillating infinite plate.

**Couette-type damper:**

$$Q_{cd} = \frac{d}{\mu A} \sqrt{MK} \quad (8-20)$$

**Q: quality factor,  $Q = \omega_n M / b$**

**Stokes-type damper:**

$$\frac{1}{Q_s} = \frac{1}{Q_{sd}} + \frac{1}{Q_{s\infty}} + \frac{1}{Q_{sc}} \quad (8-21)$$

**total=(finger to ground) + (finger to  $\infty$ ) + (finger to finger)**

where  $Q_{sd} = Q_{cd} \frac{(\cosh 2\beta d - \cos 2\beta d)}{\beta d (\sinh 2\beta d + \sin 2\beta d)}$  (8-22)

$$Q_{s\infty} = Q_{cd} \frac{1}{\beta d} \quad (8-23)$$

$$Q_{sc} = Q_{cd} \frac{A_{finger}}{A} \frac{(\cosh 2\beta z - \cos 2\beta z)}{\beta d (\sinh 2\beta z + \sin 2\beta z)} \quad (8-24)$$

where

$$\beta = \sqrt{\frac{\omega}{2\nu}}, \quad \omega: oscillation frequency, \nu: dynamic viscosity$$

$$z: finger gap, Q: quality factor \equiv \frac{\omega M}{b}$$

## ◆ Dynamic Response

**Laterally driven:**

$$M\ddot{X} + b\dot{X} + K(X - X_0) = F_{x,C} = \frac{1}{2} N \epsilon_0 V^2 \frac{W}{z} = \text{cons} \tan \iota$$

(8-25)

### Gap closing:

$$M\ddot{z} + b\dot{z} + K(z - z_0) = F_{z,C} = \frac{-1}{2} N_z \left( \frac{k\epsilon_0 W X}{z^2} \right) V^2$$

(8-26)

### Frequency response

$$\omega = \sqrt{\frac{K}{M}} \quad (8-27)$$

#### Reference

- [1] William C. Tang, Tu-Cuong H. Nguyen, and Roger T. Howe, "Laterally Driven Polysilicon Resonant Microstructures", Proceedings of MEMS '89, pp.187-193, Feb., 1989.
- [2] T. Hirano, T. Furuhata, K. J. Gabriel, and H. Fujita, "Operation of Sub-micron Gap Electrostatic Comb-drive Actuators", Proceedings of MEMS'91, pp. 873-876, Feb. 1991.
- [3] Reid A Brennen, Martin G. Lim, Albert P. Pisano, and Alan T. Chou, "Large Displacement Linear Actuator", Technical Digest IEEE Solid-State Sensors and Actuators, pp. 135-139, June, 1990.
- [4] Young-Ho Cho, Albert P. Pisano, and Roger T. Howe, "Viscous Damping Model for Laterally Oscillation Microstructures", Journal of MEMS, Vol. 3, No. 2, pp. 81-86, June, 1994.